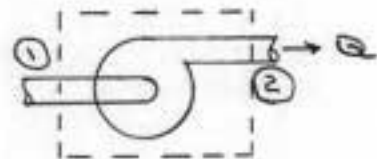


Given: Centrifugal water pump operating under conditions as follows:



$D_1 = D_2 = 4 \text{ in.}$ $Q = 300 \text{ gpm}$

$P_1 = 8 \text{ in Hg (vacuum)}, P_2 = 35 \text{ psig}$ $z_1 = z_2$

$P_{\text{input}} = 9.1 \text{ hp}$

Find: pump efficiency.

Solution: Apply the energy equation to the cv shown. Neglect all losses to find the energy added to the fluid.

Basic equations: $\eta = \frac{\dot{w}_s}{P_{\text{in}}}$ where $\dot{w}_s = \text{power into fluid}$

$$\dot{Q} - \dot{w}_s - \dot{w}_{\text{shear}} - \dot{w}_{\text{other}} = \frac{\partial}{\partial t} \int_{cv} \rho p \, dV + \int_{cs} \left(u + Pv + \frac{V^2}{2} + gz \right) \rho \vec{v} \cdot d\vec{A}$$

- Assumptions:
- (1) $\dot{Q} = 0$
 - (2) $\dot{w}_{\text{shear}} = 0$ (by choice of cv); $\dot{w}_{\text{other}} = 0$
 - (3) steady flow
 - (4) neglect Δu
 - (5) $\Delta z = 0$
 - (6) incompressible flow
 - (7) uniform flow at inlet and outlet

Then $-\dot{w}_s = (P_1 V_1 + \frac{V_1^2}{2}) \dot{m} + (P_2 V_2 + \frac{V_2^2}{2}) \dot{m}$

Since $\dot{m} = \rho Q$ and $V_1 = V_2$ (from continuity)

$-\dot{w}_s = \rho Q (P_2 V_2 - P_1 V_1) = Q (P_2 - P_1)$

$P_1 = \rho g h = 5.2 \rho_{\text{H}_2\text{O}} g h$

$P_1 = 13.6 \cdot 1.94 \frac{\text{slug}}{\text{ft}^3} \cdot 32.2 \frac{\text{ft}}{\text{s}^2} \cdot (-8 \text{ in}) \cdot \frac{\text{ft}}{12 \text{ in}} \cdot \frac{\text{lb} \cdot \text{s}^2}{\text{ft} \cdot \text{slug}} \cdot \frac{\text{ft}^2}{144 \text{ in}^2} = -3.93 \text{ psig}$

$\therefore -\dot{w}_s = 300 \frac{\text{gal}}{\text{min}} \cdot \frac{\text{ft}^3}{7.48 \text{ gal}} \cdot \frac{\text{min}}{60 \text{ s}} \cdot [35 - (-3.93)] \frac{\text{lb}}{\text{in}^2} \cdot \frac{144 \text{ in}^2}{\text{ft}^2} \cdot \frac{\text{hp} \cdot \text{s}}{550 \text{ ft} \cdot \text{lb}}$

$\dot{w}_s = -6.81 \text{ hp}$ (negative sign indicates energy added)

Then

$\eta = \frac{\dot{w}_s}{P_{\text{in}}} = \frac{6.81}{9.1} = 0.748 \text{ or } 74.8 \text{ percent}$